

Constraint-Based Contact Analysis between Deformable Objects

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Abstract. The key to the successful simulation of deformable objects is to model the realistic behavior of deformation when they are influenced by intricate contact conditions and geometric constraints. This paper describes constraint-based contact modeling between deformable objects using a nonlinear finite element method. In contrast to the penalty force based approaches, constraint-based enforcement of contact provide accuracy and freedom from finding proper penalty coefficients. This paper is focused on determining contact regions and calculating reaction forces at appropriate nodes and elements within the contact regions. The displacement and deformation of all nodes are dynamically updated based on the contact reaction forces. Our constraint based contact force computation method guarantees tight error bound at the contact regions and maintains hard constraints without overshoot or oscillation at the boundaries. In addition, the proposed method doesn't require us to choose proper penalty coefficients, thus greater numerical stability can be achieved and generally large integration steps can be utilized for the ODE solver. Contact conditions are formulated as nonlinear equality and inequality constraints and the force computation is cast into a nonlinear optimization problem. Our rigid-to-deformable and deformable-to-deformable contact simulation demonstrates that the non-penetration constraints are well maintained.

1 Introduction

With the increased demand for the visual realism in character animation, medical and scientific visualization, deformable object simulation is becoming a major issue. In medical visualization, for example, realistic deformation is quite complex to achieve where skin, muscle, ligaments and organs are all highly deformable and in constant contact. Thus far, deformable object simulation and animation have been addressed

from the modeling perspective where the main focus was to accurately and efficiently represent the deformation itself based on the given external forces [8]. Previous research demonstrated how deformable bodies are reacting and influenced by the known external forces, ignoring complex contact interplay between multiple deformable objects [1,11,17]. However, deformable object simulation should be addressed in a broad context of interaction with surrounding environments such as surface contacts and geometric constraints [14]. When two flexible objects collide, they exert reaction forces on each other resulting in the deformation of both objects. While many important breakthroughs have been made in modeling the deformation, the simulation of isolated deformable objects without an accurate contact model has few practical applications. Baraff et al. [2] have presented a flexible-body model that represents a compromise between the extremes of the nodal and rigid formulations and demonstrated the dynamic simulation of flexible bodies subject to non-penetration constraints. However, their flexible bodies are described in terms of global deformations of a rest shape and limited to specific geometric structures.

Also previously, a simplified particle system [11] and the depth field method [20] have been used to simulate the contact. Joukhadar et al. [19] demonstrated a fast contact localization method between deformable polyhedral. Hirota et al. [18] used penalty forces to prevent self-collision of FEM-based human model. Baraff [4] also implemented similar spring types of penalty force to prevent inter-penetration for cloth simulation. However, these approaches allow penetration upfront and measure the penetration depth to estimate the force to rectify the situation. These approaches are prone to the overshoot and oscillation problem if the stiffness coefficient is not correct, and the larger coefficients make the integration stepping suffer. However, an even more serious problem is that determining proper values for the stiffness of the spring is not trivial. Therefore, sliding contact between two deformable objects is very problematic while using a penalty-based method to maintain a tight error bound. Recent researches in cloth simulation using repulsions, penalty force and geometric fixes [5, 10, 15] also share similar problems.

This paper demonstrates a computational scheme for representing the geometry and physics of volumetric deformable objects and simulating their displacements when they are colliding and preserving accurate contact positions. The proposed technique focuses on determining contact regions of both objects and calculates accurate contact reaction forces at appropriate nodes and elements to maintain the resting contact relations. Our penetration-free deformation is based on the consolidation of colliding and resting contacts often used in rigid body simulation [3] and uses quadratic programming [21] to solve the non-penetration constraints.

2 Collision Detection and Contact Region Identification

Our deformation model is based on the standard nonlinear finite element method (FEM) analysis [6, 7] with Cauchy-Green deformation tensor and we have chosen to use a nearly incompressible Neo-Hookean material [9]. Since deformable object is discretized with a finite number of elements, the collision detection problem can be

interpreted as determining minimum distance between two non-convex boundary polyhedra. Collision detection has attracted considerable attention in geometric modeling and robotics [12]. A volumetric object is meshed with a set of tetrahedra and the surface is represented with a set of triangles. An entire object is approximated with a hierarchical tree of axis-aligned bounding boxes (AABB) to facilitate the quick collision rejection. Although an AABB quick reject test eliminates substantial computation time, actual costly intersection tests between geometric primitives at the leaves of the trees are inevitable. We are mainly interested in vertex-face collision and with an option to turn on and off edge-edge collision detection. This issue is detailed in the contact force computation section.

Once the penetrated objects and the intersections are found, the simulator must compute the exact collision time to include contact forces to prevent further penetration. Finding the exact collision time is time consuming [13], so often a numerical estimate is preferred within a certain tolerance. In addition, finding the exact collision time between deformable bodies using back-tracking and binary search is not practical because, unlike rigid objects which have relatively few nodes per object, soft objects contact may include numerous nodes that can penetrate the surfaces of other objects in a given time step. Thus potentially vast number of triangle-edge collisions can occur repeatedly during back-tracking and simulation can be slowed down substantially. Instead, when a penetration is detected between a node and a surface, the inter-penetrated nodes are moved back onto the surface. By doing so, we use a completely inelastic collision, similar to a zero restitution coefficient, from the colliding node's point of view. The artificial abrupt deformation is estimated and added to the node's force vector by interpolating internal forces of the colliding element based on the distance it is moved back. Then, the simulator sets up relative velocities of the node and surface contact points (computed with 3 nodes of the surface) to estimated velocities which are calculated by interpolation of relative velocities to ensure the legal condition of the collision for a non-penetration constraint. Changes in the internal force of the colliding node represent the compression and stored energy of the colliding elements and subsequently used for expansion and separation. This process simplifies the collision detection so that the actual contact force computation can be done more efficiently.

3 Contact Force Computation

Once the collisions are detected and the penetrated feature list is formed, the contact constraints are formulated and reaction forces are computed to prevent the penetration. Our collision and contact model is an extension from well-known rigid body polygonal objects contact scheme [3] to deal with multiple contact points within the contact region and the management of active constraint set. As fig. 1 illustrates, at a given time step, a point x on the contactor surface (Body I) can be either in contact with a point y on the target surface (Body J) or outside the surface. This can be verified by calculating the distance $g(x, y)$ from x to y . If $g(x, y) > 0$, there is no contact

between the two points and the normal contact force λ between these points is equal to zero. On the other hand, if $g(x, y) = 0$, the two points are in contact and the contact force has a non-zero value. These complimentary conditions can be shown as follows:

$$g(x, y) \geq 0, \lambda \geq 0, g\lambda = 0$$

To model the contact relation between deformable bodies, above conditions should be applied at each of the contact nodal points.

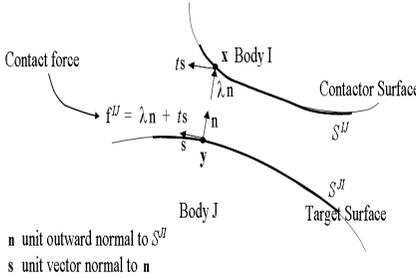


Fig. 1. Contact conditions

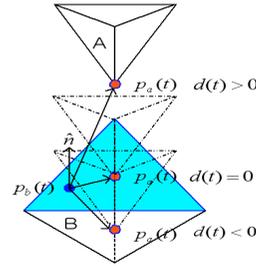


Fig. 2. Contact conditions between two tetrahedral

Our computation model for constraint-based contact analysis focuses on converting the contact cases into non-penetration constraints and formulates the contact forces to maintain the constraints at the contactor and target nodal points. Contact forces are applied exactly the same as external forces and generate deformation on both sides. Fig. 2 shows three possible contact scenarios, before, at contact, and after the penetration between two tetrahedra with a vertex-face contact. When the two bodies are disjoint at a given time t_0 , the prospect contact points (closest point to the other object) on each side of object P_a and P_b are separated and the relative distance d is positive. However, when an actual penetration happens, the relative distance becomes negative as illustrated in fig. 2. $P_a(t_0) = P_b(t_0)$ denotes that two points are in contact at time t_0 . The contact status of two points at time t_0 can be written in distance function in the normal direction as:

$$d_i(t_0) = \hat{n}_i(t_0) \cdot (P_a(t_0) - P_b(t_0))$$

where \hat{n}_i is the outwards unit surface normal of a point i on the surface of object B. [0]Once the collision detection routine finds all colliding elements, non-penetration constraints are formulated to prevent further penetration in the next time step. Our simulator enforces this constraint by setting the relative velocity to zero and by maintaining the relative acceleration to be greater than or equal to zero. The relative velocity $\dot{d}_i(t)$, a time derivative of $d_i(t)$, can be represented as:

$$\dot{d}_i(t) = \dot{\hat{n}}_i(t) \cdot (P_a(t) - P_b(t)) + \hat{n}_i(t) \cdot (\dot{P}_a(t) - \dot{P}_b(t))$$

Starting from $d_i(t) = 0$ and having $\dot{d}_i(t) = 0$ ensures that the two points are not moving toward penetration or separation at the current time step. Relative acceleration with respect to the nodal positional acceleration can be written as:

$$\ddot{d}_i(t) = \ddot{\hat{n}}_i(t) \cdot (P_a(t) - P_b(t)) + 2\dot{\hat{n}}_i(t) \cdot (\dot{P}_a(t) - \dot{P}_b(t))$$

To prevent inter-penetration, the condition $\ddot{d}_i(t) \geq 0$ must be maintained. Since $\ddot{d}_i(t)$ has nodal positional accelerations $\ddot{P}(t)$ term for both side of objects, and nodal accelerations can be rewritten with repulsive outward contact force $f_{ci}(t) \geq 0$ and mass.

Fig. 2 only illustrates the vertex-face case but edge-edge case is similar. Once the collision is detected, there are two contact points at each side of an edge and the normal direction can be computed with a cross product of two edges to represent the perpendicular direction. Computing the relative velocity and acceleration is similar to the vertex-face case. Since we're using a mesh structure to discretize a volume, the density of mesh determines the overall resolution and details of objects. If the average unit tetrahedron size is small enough, then the edge-edge case can be ignored since the edge-edge penetration can be detected by the vertex-face within a few times. However, if the size of average tetrahedron is relatively big, then the edge-edge penetration shows a significant visual artifacts and it must be removed. For deformable objects where each volumetric structure is meshed with fine triangles and tetrahedra, enforcing the edge-edge condition often causes additional computing load without substantial improvement in visual realism. In addition, sometimes enforcing edge-edge condition make the two objects remain in contact on an edge instead of a contact area, resulting local separation gaps within the contact area. Our system adopts an option to ignore the edge-edge cases when the mesh size is relatively small compare to the size of object volume.

4 Numerical Solution Method

To maintain the contact constraint, contact force must be positive to represent repulsive force toward the outside of the contact face $f_{ci}(t) \geq 0$ or the relative acceleration must be $\ddot{d}_i(t) \geq 0$ to guarantee the object separation. In addition, if any one of those conditions is met, then the other inequality must be zero. This last condition can be written as $f_{ci}\ddot{d}_i(t) = 0$. These complementary conditions can be arranged as a quadratic programming problem [21] with a general form as follows:

$$\begin{aligned} &\text{Minimize} && c^T x + \frac{1}{2} x^T Q x \\ &\text{Subject to} && Ax \geq b, \quad x \geq 0 \end{aligned}$$

Since the function $f_{ci}\ddot{d}_i(t) = 0$ is monotonic, the minimization will have at least one solution and will converge to zero. Therefore, it can be used as an objective function and the $f_{ci}(t) \geq 0$ and $\ddot{d}_i(t) \geq 0$ conditions can be considered as inequality constraints in the QP system. If we rewrite the relative acceleration as a function of unknown contact forces as:

$$\ddot{d}_i(t) = a_{i1}f_{c1} + a_{i2}f_{c2} + \dots + a_{in}f_{cn} + b_i$$

where a_{ij} is a n by n matrix for coefficients of unknown contact force, then we can get the linear and quadratic coefficients c^T and Q . Interior Point Method for QP often uses slack variables to make all inequality constraints into non-negatives. Non-negativity constraints are replaced with logarithmic barrier terms and added to the objective function. As the non-negativity constraints approach zero, the logarithmic barrier function becomes negative infinity, moving the objective function to the positive infinity. Therefore, the objective function will prevent them from becoming negative. The remaining linear constraints are all equalities and Lagrangian multipliers can be applied to get the local minima. The condition for the local minima is Karush Kuhn Tucker (KKT) condition. Incorporating the equality constraints into the objective function using Lagrange multipliers makes the system:

$$\text{Minimize } c^T x + \frac{1}{2} x^T Q x - \mu \sum_i \log x_i - \mu \sum_i \log w_i + \bar{\lambda}^T (b - Ax + w)$$

where μ is a weight coefficient. If we take the derivative of the objective function and set it to zero for the local minima, we then have a resulting system of $2m + 2n$ linear equations with $2m + 2n$ unknowns, where m and n denote the number of nodes and constraints. Since this is sparse linear system, we can apply well-known sparse linear system solvers including conjugated gradient method or Cholesky factorization.

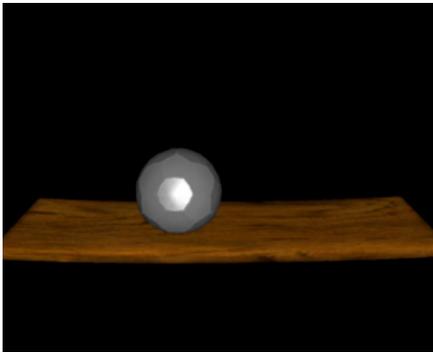


Fig. 3. Ball and board in contact

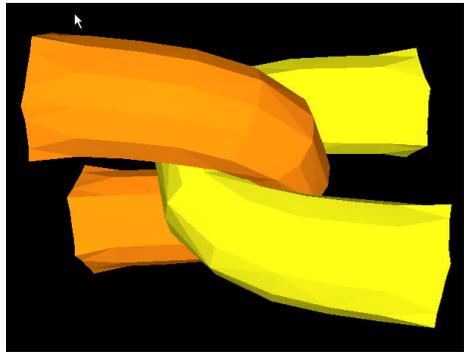


Fig. 4. Two deformable cylinders in contact

5 Experiments

We have described a FEM-based deformation simulator and implemented collision detection and contact resolution modules. Our simulator uses the 4th-order Runge-Kutta ODE solver with adaptive step sizes. Fig. 3 shows a contact simulation between two deformable bodies, a stiff wood board and another stiff heavy metal ball. An animation [16] shows its accurate maintenance of the contact condition as well as realistic behaviors of the two stiff objects under contact. Since the models have relatively fine mesh structure, the simulation was performed and rendered off-line. Fig. 4 shows two deformable cylinders in contact. These cylinders are relatively coarsely

meshed and softer than the example shown in fig. 3. This example demonstrates that the contact and overall volumes of the objects are preserved even if they are under large deformation. An internal view from the animation shows the cylinders wrap around each other shows no penetrated nodes under the large deformation. The accuracy of our collision and contact model is quantitatively evaluated by measuring penetration and/or separation distance at contact regions.

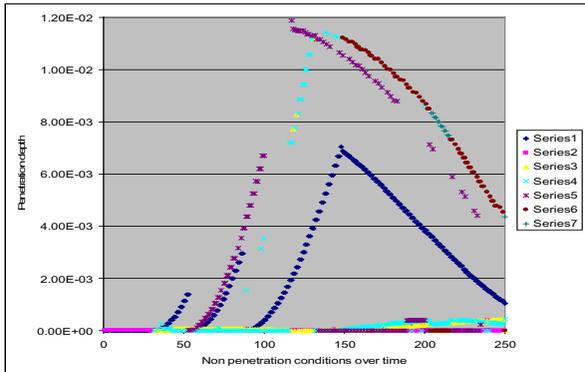


Fig. 5. Penetration depth

Fig. 5 shows an experimental penetration depth data from the simulation of two deformable bodies in contact shown in fig. 4. It runs in real-time and programmed to record all contacts error measures. It sampled 7 areas of contacts with the total of about 250 individual contact constraints enforced over time. The tolerance of penetration is set to $10E-2$ and the average unit length of an element is 10, so it allows about $1/1000$ of the size of a tetrahedral element for legal penetration. As shown in the graph, a majority of contact constraints are maintained at $10E-3$ precision. Some contacts undergo penetrations up to the tolerance level of $10E-2$ but nonetheless they are controlled within the user-defined error bound. Although the stability of simulation is heavily dependent to the material properties and ODE step sizes, our QP solver converges well within the pre-defined error bounds. For models with approximately 500 tetrahedral elements each side, our simulator performs at a visual interactive rate with an average of 25 frames per second on a Pentium 4 1.7GHz.

6 Conclusion and Future Works

This paper described a constraint-based collision and contact simulation between deformable bodies. The hierarchical collision detection and the initial conditioning process for non-penetration constraint enforcement simplifies the constraint formulation and accelerates the overall contact simulation. The nonlinear optimization based contact constraint enforcement demonstrates tight error bound at the contact region and numerical stability. The performance of the system is efficient enough to run medium to small scale FEM model in real-time. Our constraint-based contact elimi-

nates the needs for computing proper penalty force coefficients. Although some coarsely meshed objects can be simulated in real-time using a generic PC, deformation and contact between densely meshed structures still remains as a significant challenge. Proper parallelization of FEM deformation with respect to the collision detection, and contact force computation could be one of the natural extensions from this work. Adaptive re-meshing and simulation based on the amount of deformation and area of interest would also substantially increase the performance as well.

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